



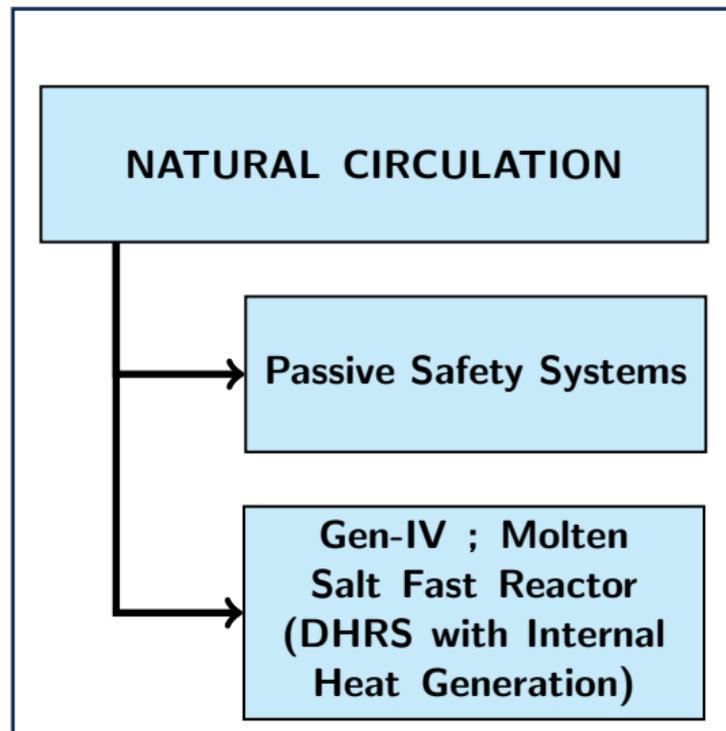
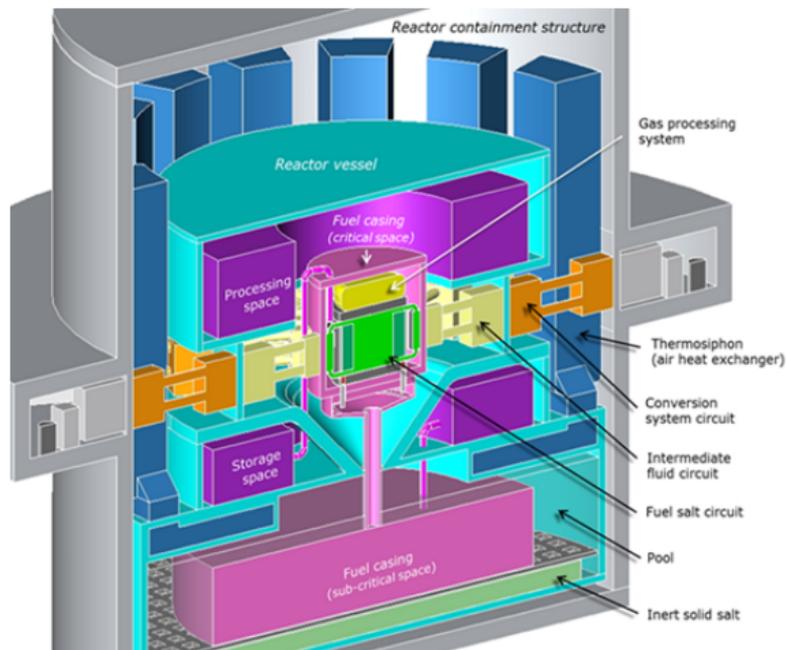
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## **MODELLING OF THE DYNASTY EXPERIMENTAL FACILITY FOR NATURAL CIRCULATION UNDER DISTRIBUTED HEATING**

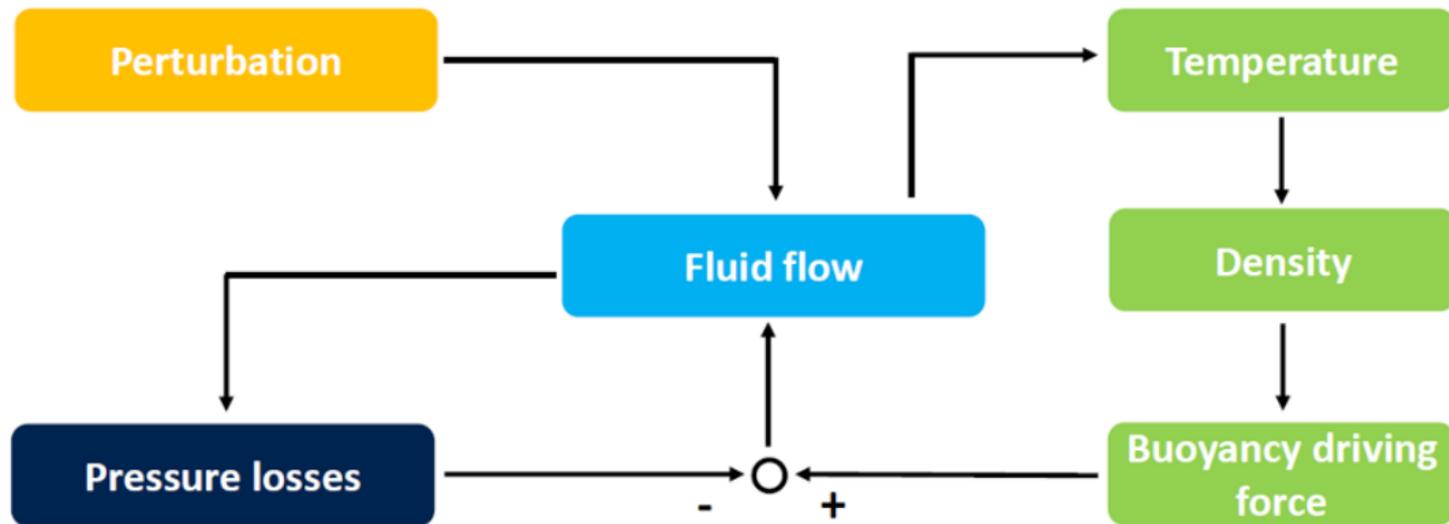
**Carolina Introini, Gabriele Benzoni, Antonio Cammi, Stefano Lorenzi**

Technical Meeting on state-of-the-art Thermal Hydraulics of Fast Reactors, 26-30/09/22



**SAMOSAFER**

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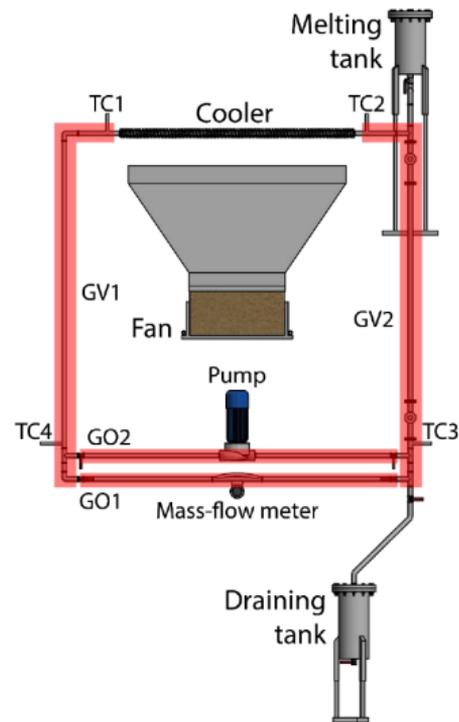
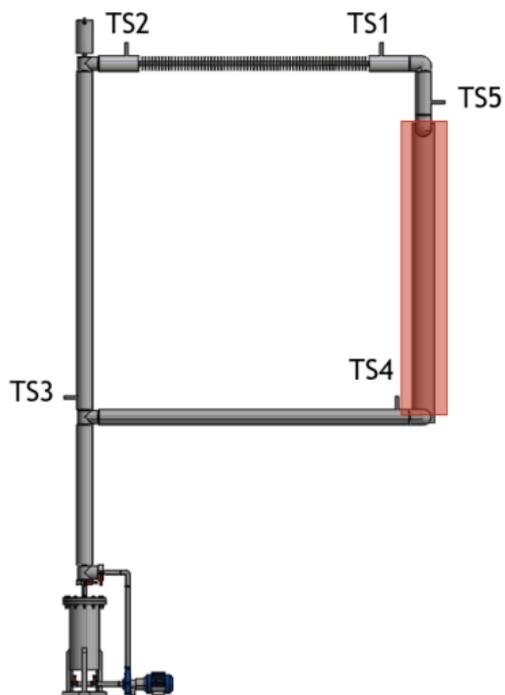
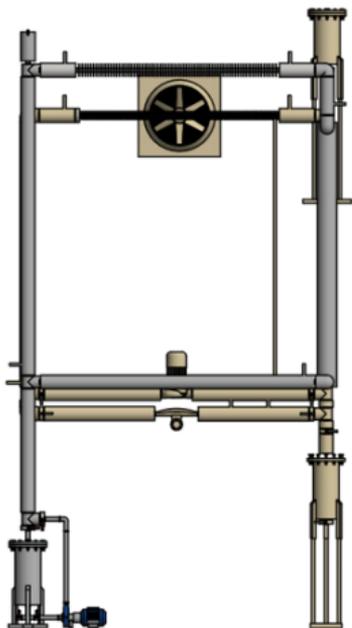
- Flow behaviour in loops is governed by density wave instabilities (Welander).
- Possible flow inversion and recirculation regions

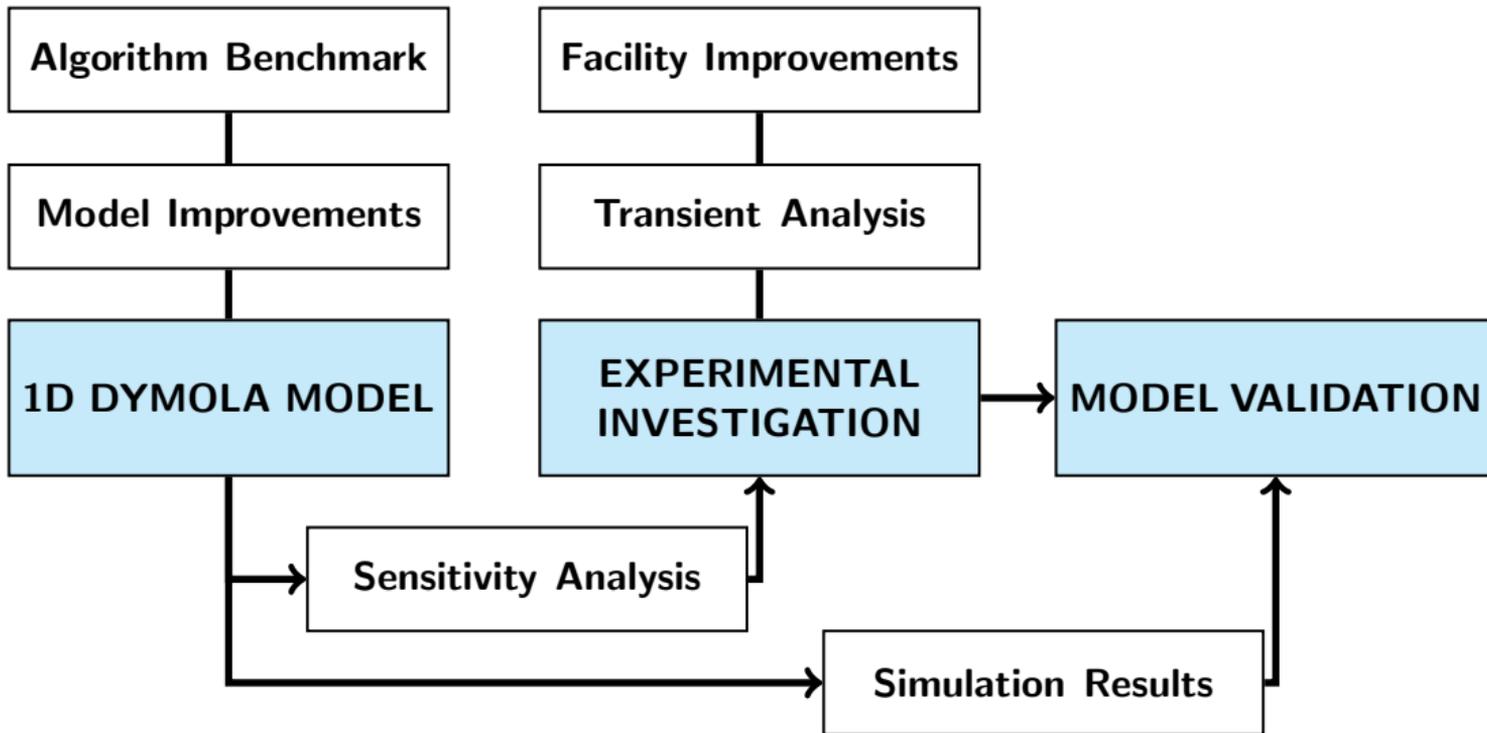


- IHG simulated through DEHS
- No insulation
- Maximum heating power per leg (step)
- Working fluid: water
- Temperature working threshold: 95 °C
- Clockwise mass flow taken as positive

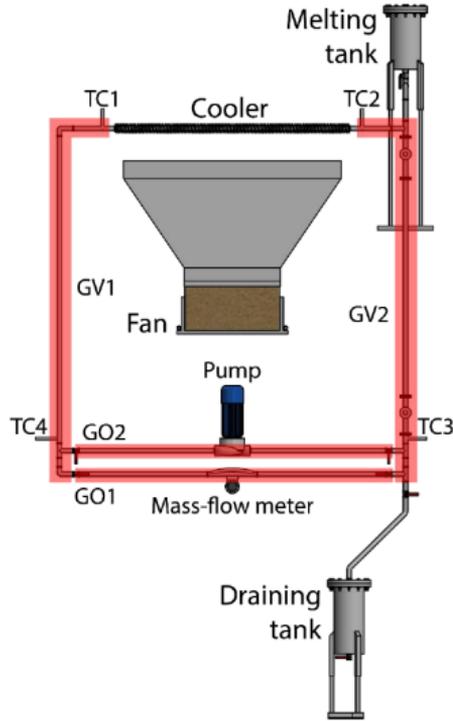
# Introduction Coupled System

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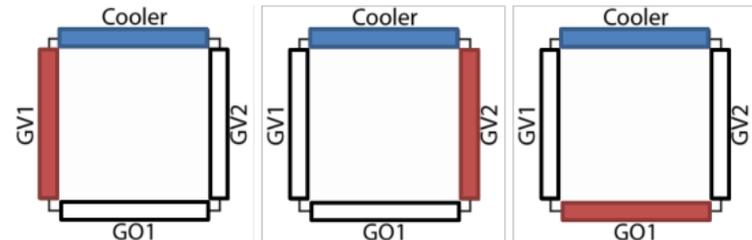


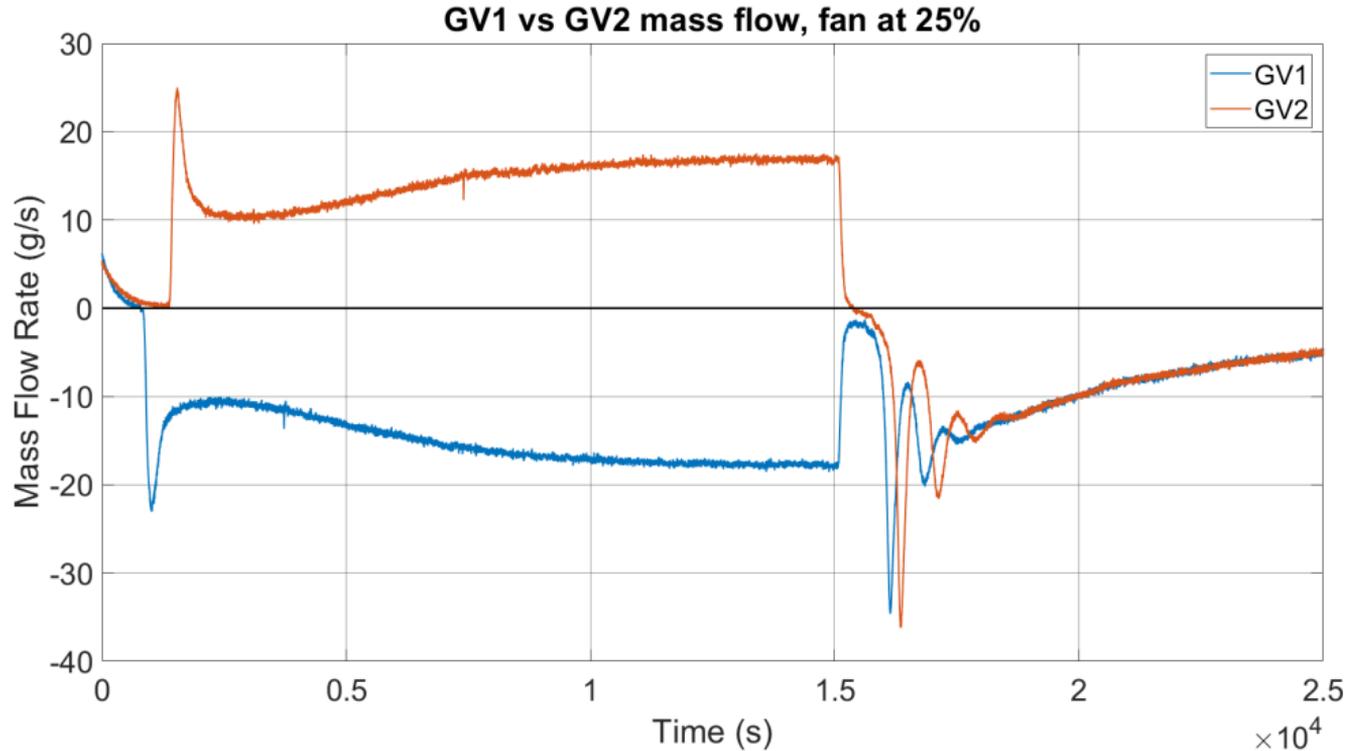
# DYNASTY Configuration

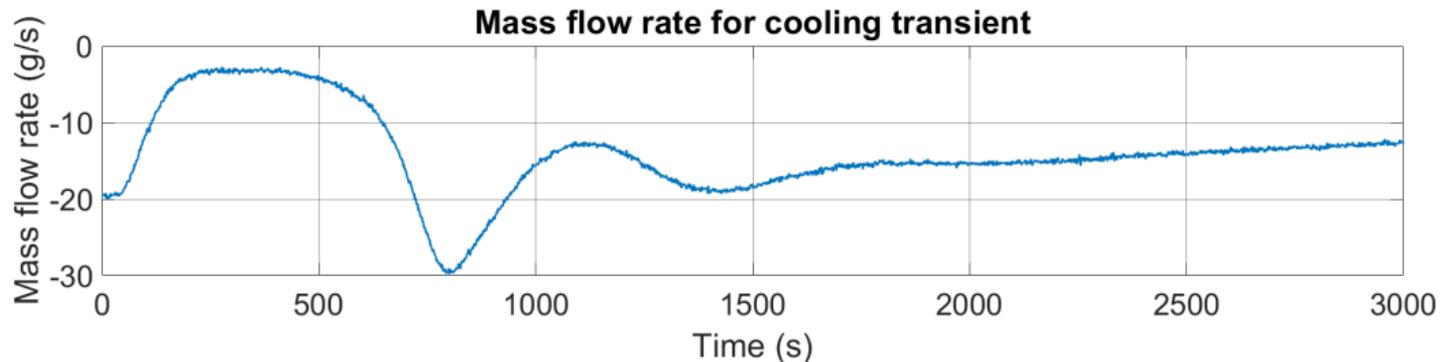
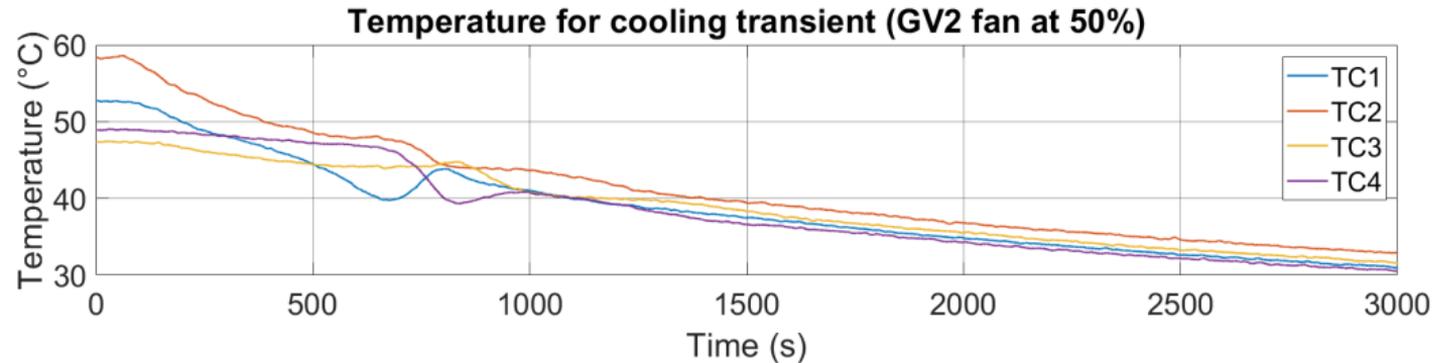


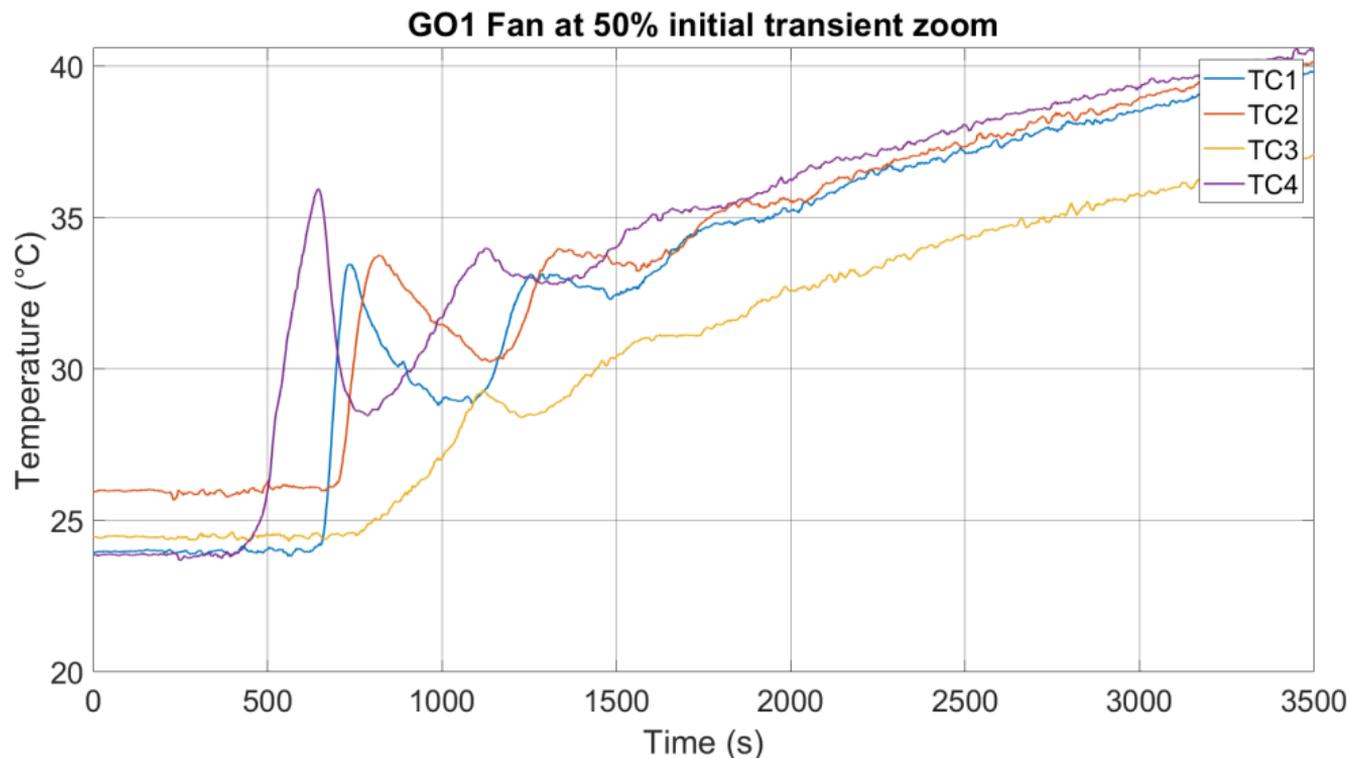
	VHHC GV1	VHHC GV2	HHHC GO1
	0%	0%	0%
	25%	25%	25%
	50%	50%	50%
	75%	75%	75%
Power	1800 W	1800 W	800 W

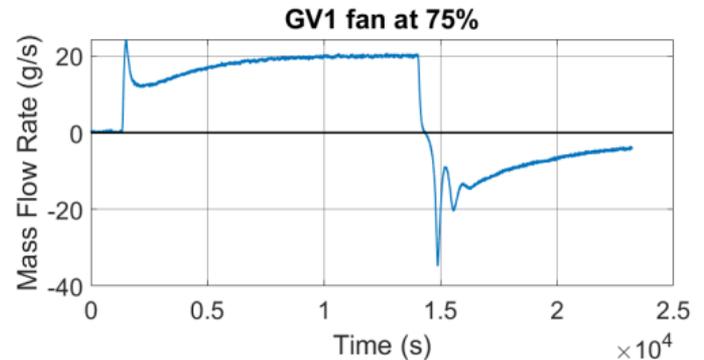
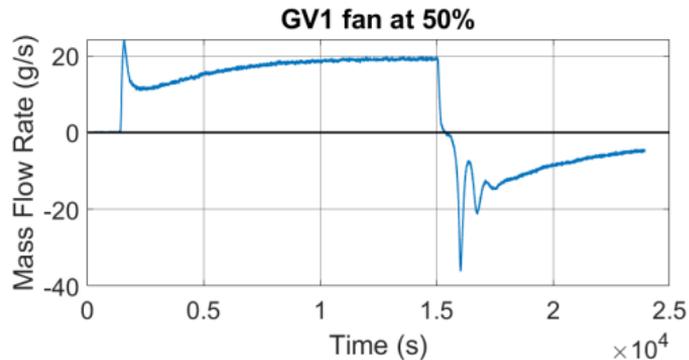
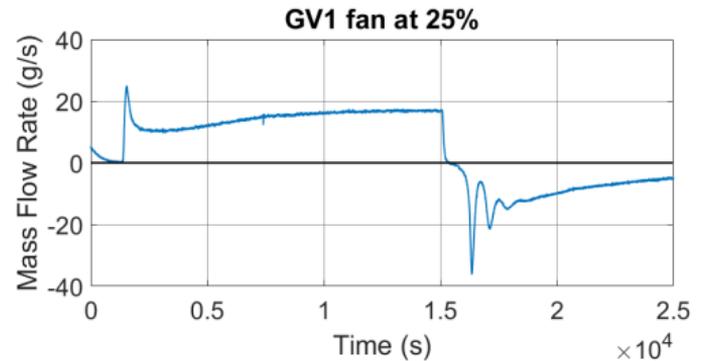
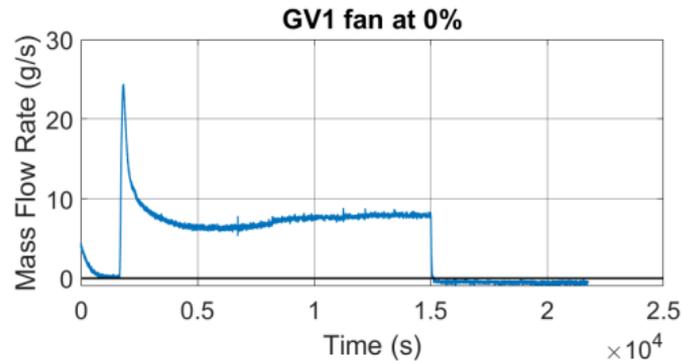
Table: Fan speed values

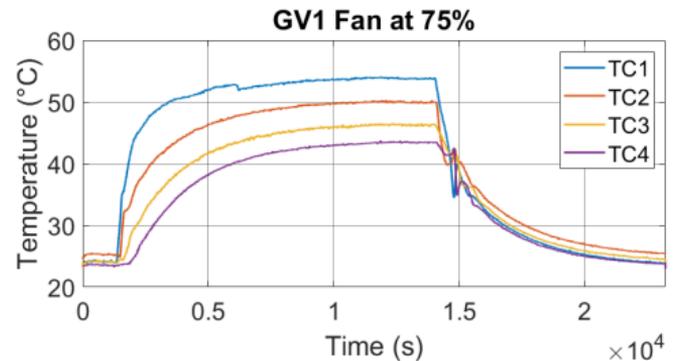
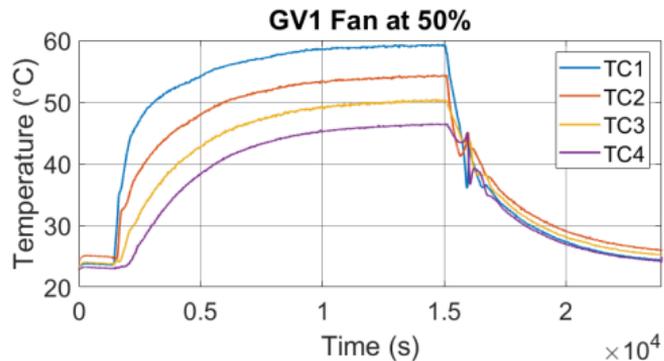
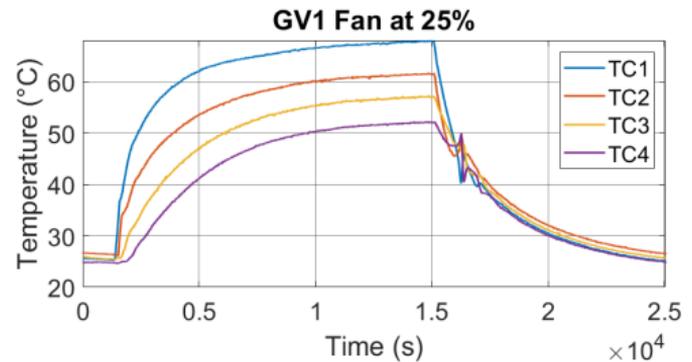
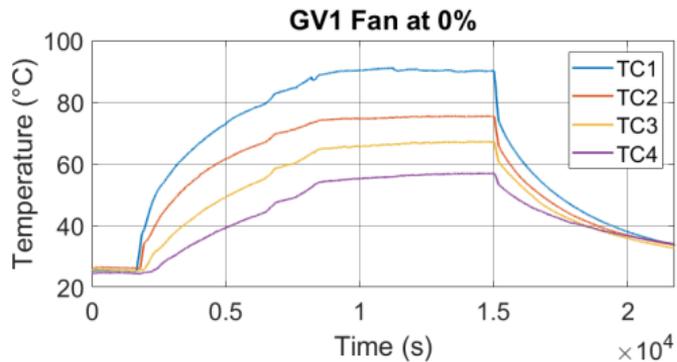


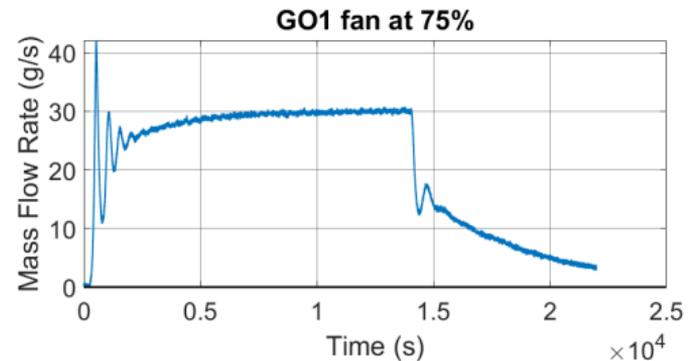
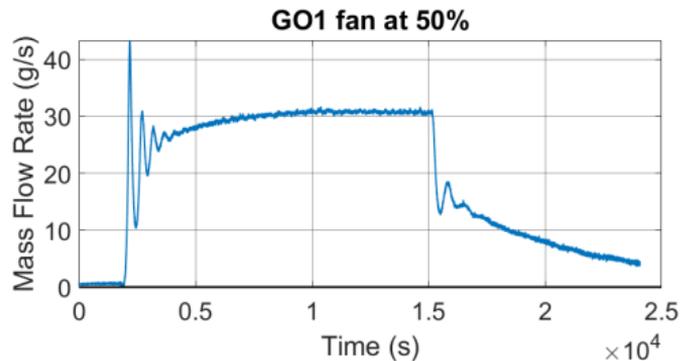
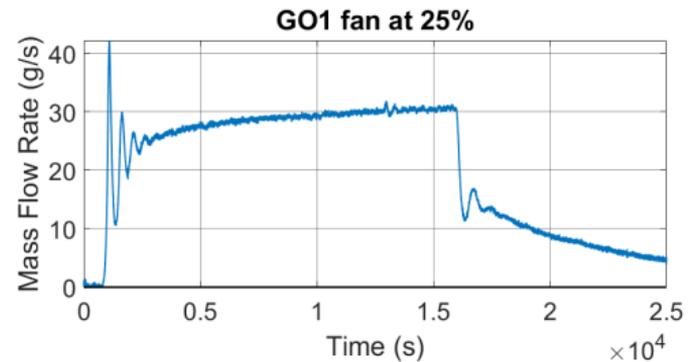
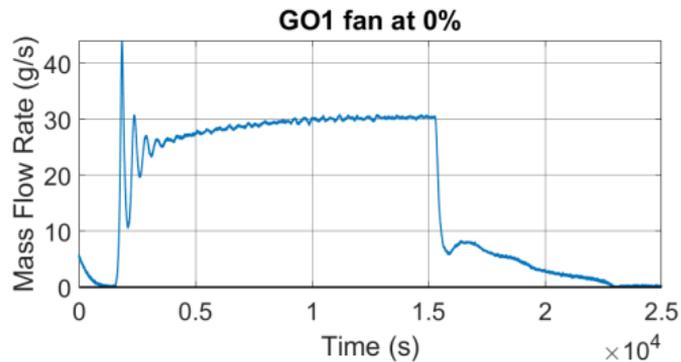


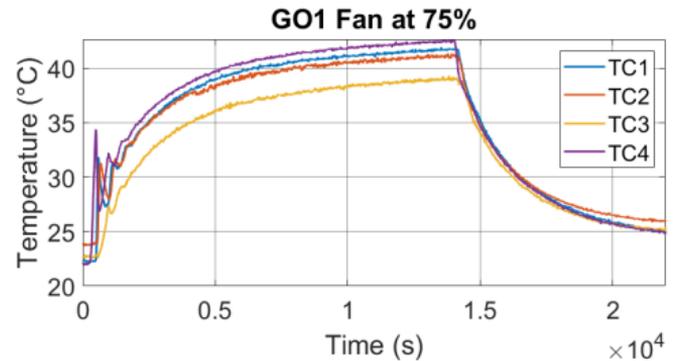
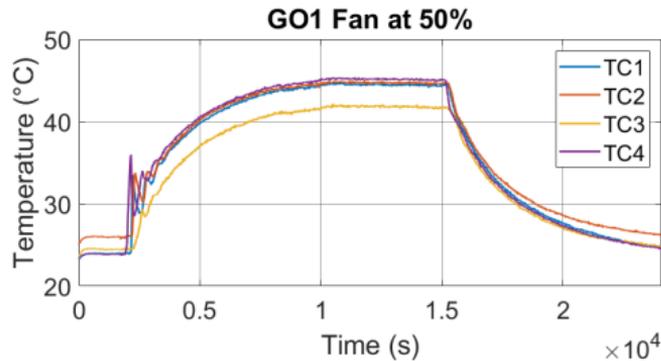
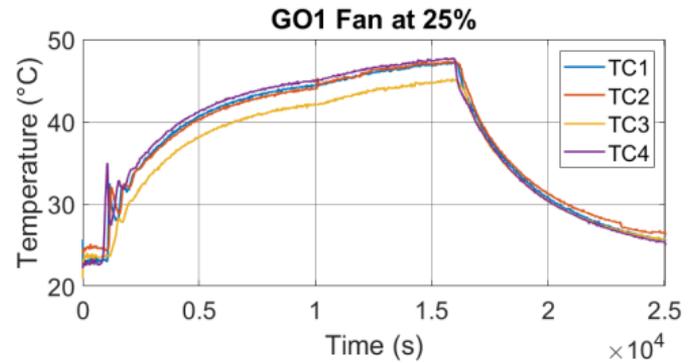
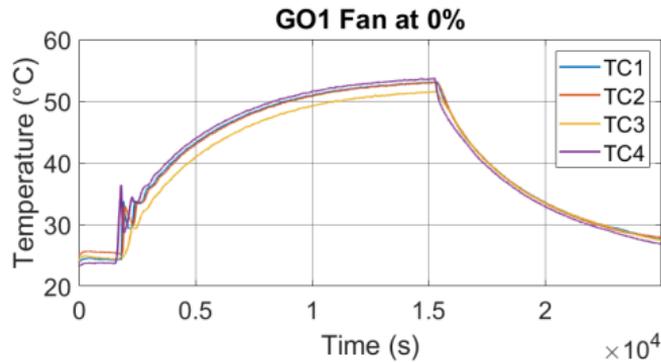


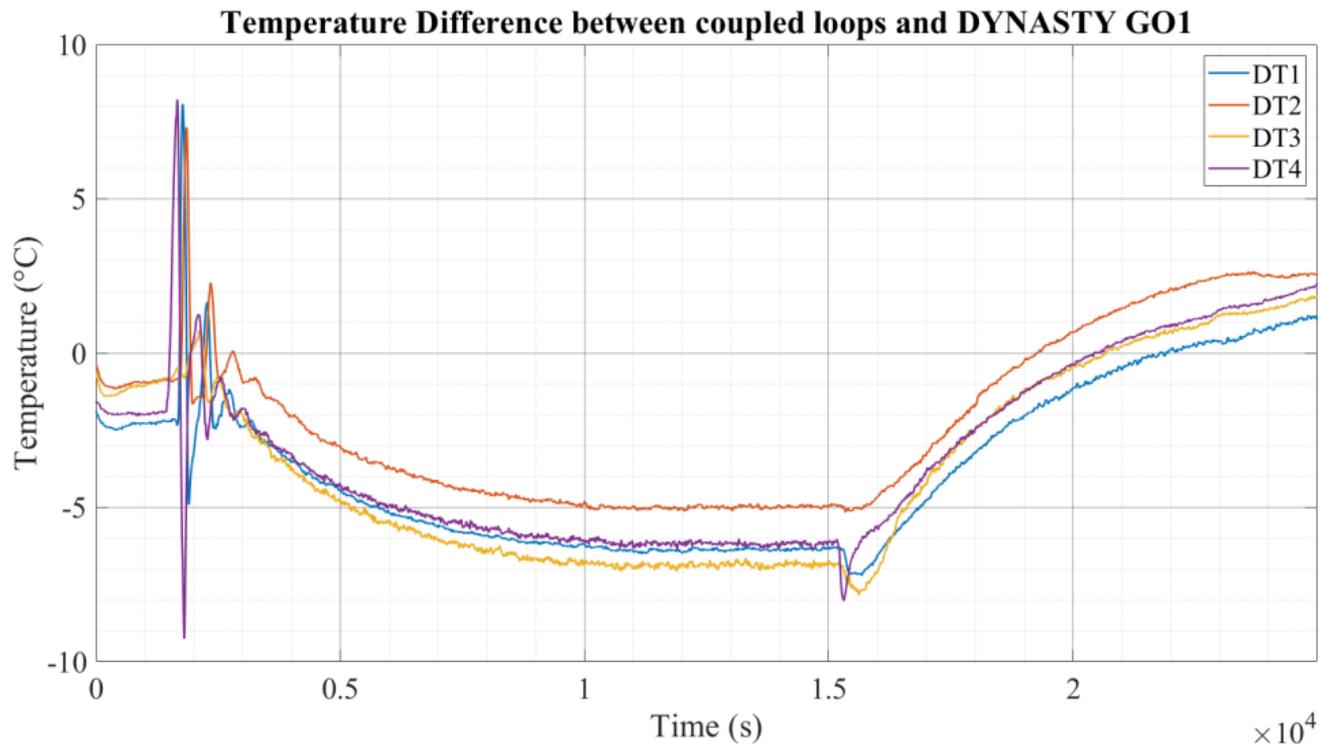


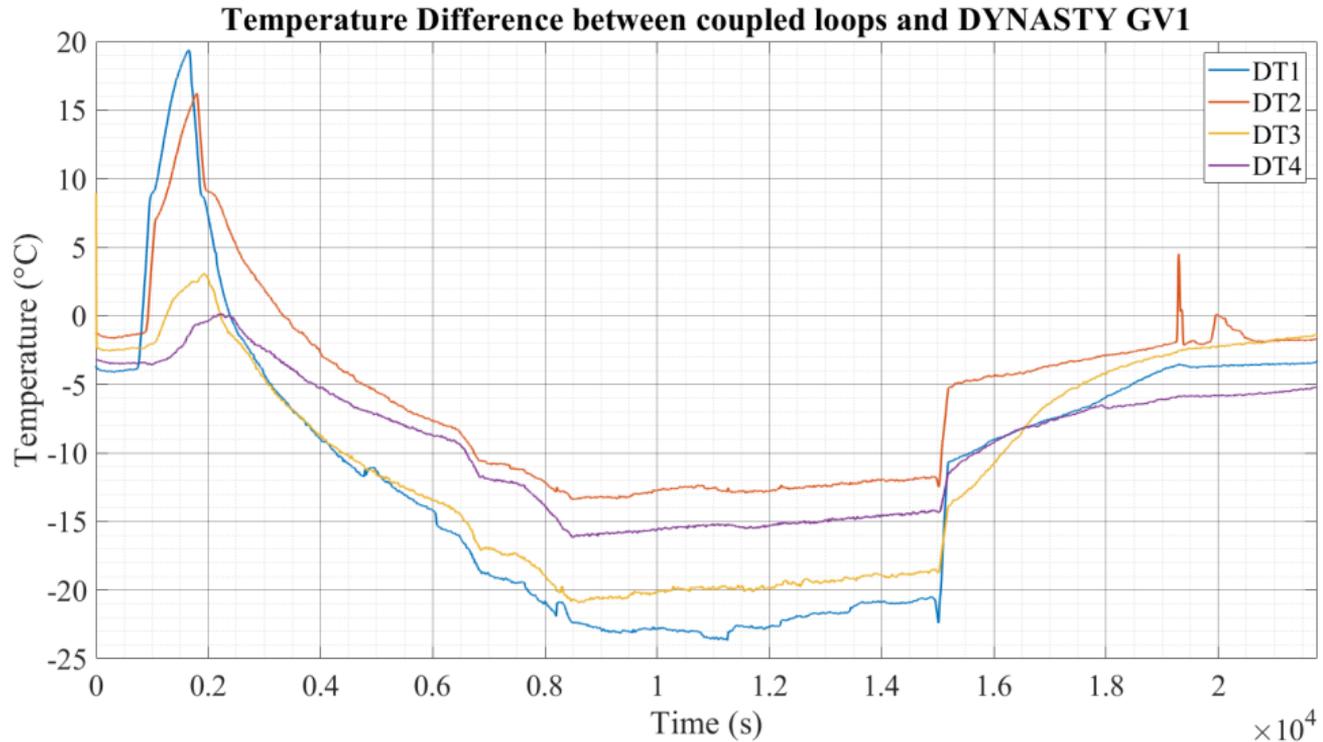






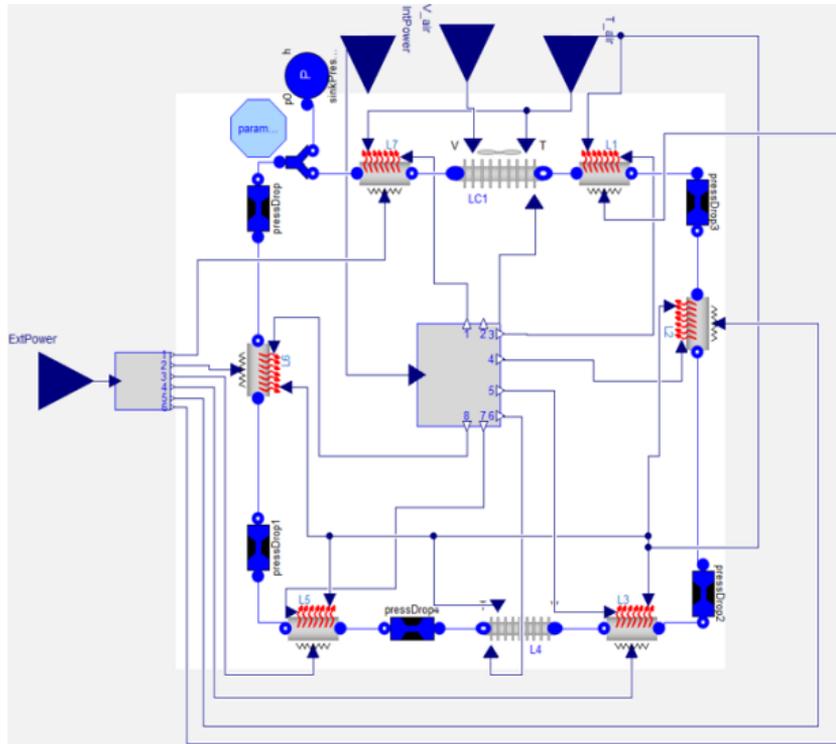






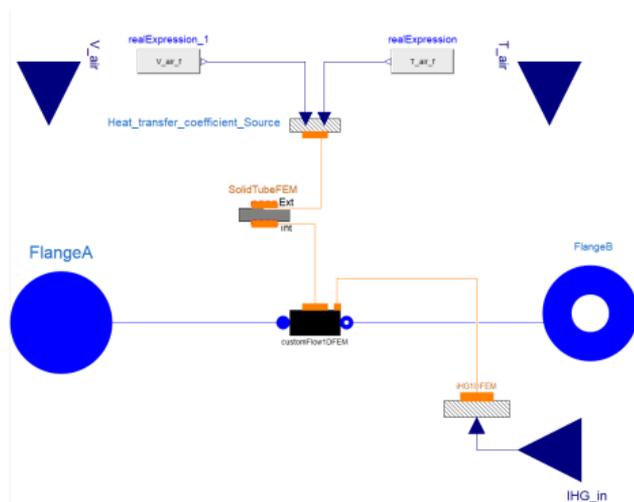
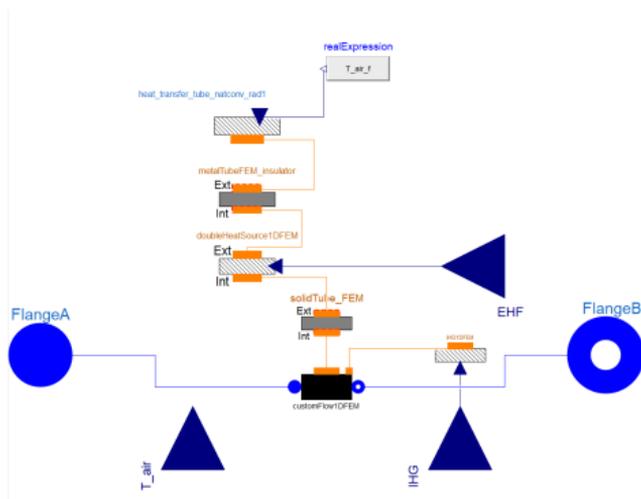
- Object-Oriented simulation language
- Acausal description using physical and engineering principles and balance equations
- Stand-alone components blocks linked by interfaces
- Differential Algebraic Equations, that must be translated into a Stiff ODE system
- Non-negligible dependency on the chosen integration algorithm.





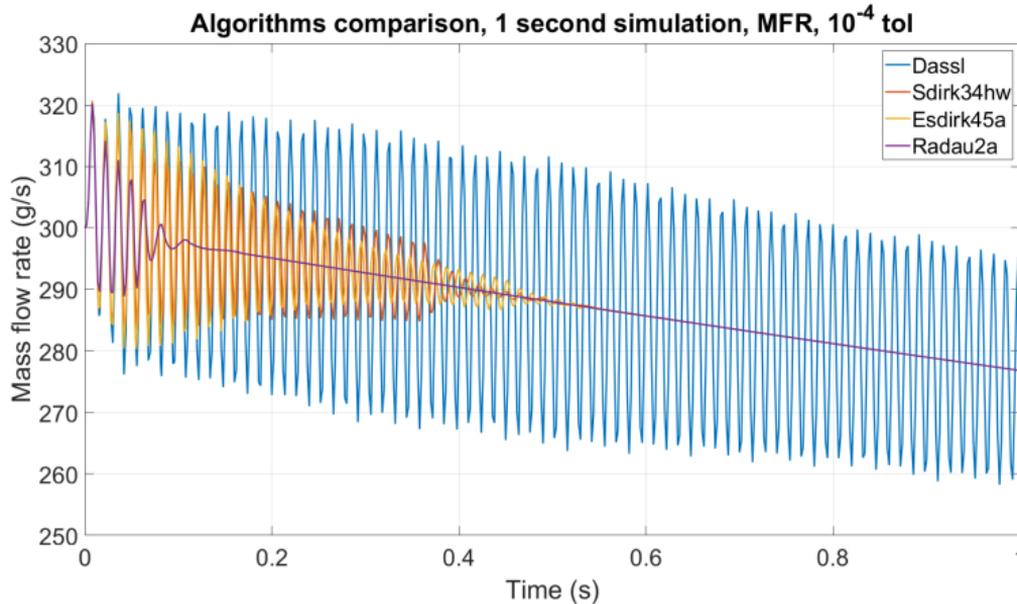
## Model upgrades

- **Heat losses** between the facility and the environment
- **Finned** cooler model also accounting for natural circulation with air
- Churchill-Bernstein correlation for  $h$
- **Pressure losses** for the MFM
- **Heat losses** for the mass flow meter (non-insulated pipe)
- Loading tank simulated imposing the pressure of the loop equal to the ambient one

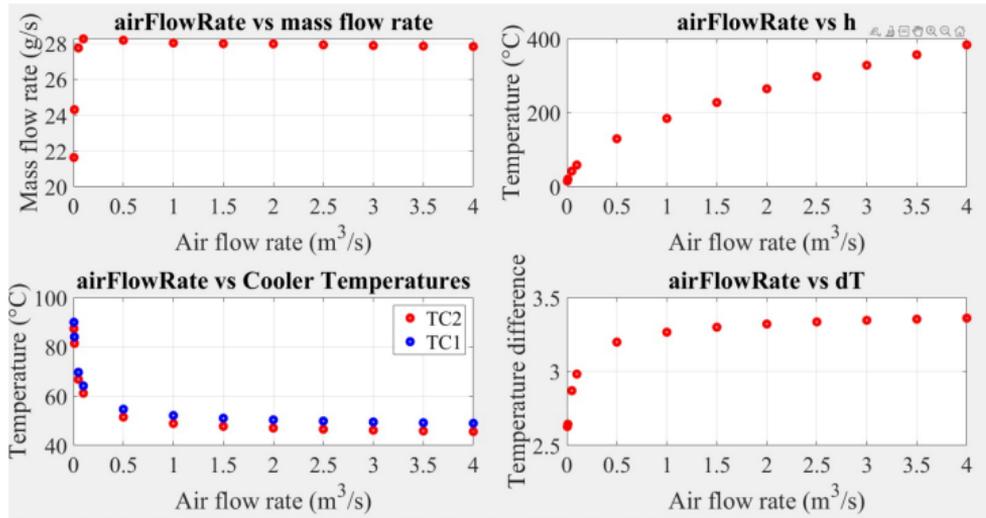


<b>Algorithm</b>	<b>Method</b>	<b>Stiff</b>	<b>Implicit</b>	<b>Order</b>	<b>Complete</b>
<b>DASSL</b>	Linear multi-step (BDF)	Yes	Yes	1-5	Yes
<b>Radau2a</b>	Single-step (RK)	Yes	Yes	5	Yes
<b>SDIRK34hw</b>	Single-step (RK)	Yes	Yes	4	Yes
<b>ESDIRK45a</b>	Single-step (RK)	Yes	Yes	5	Yes
<b>LSODAR</b>	Linear multi-step (mixed)	Yes	Both	1-5	No

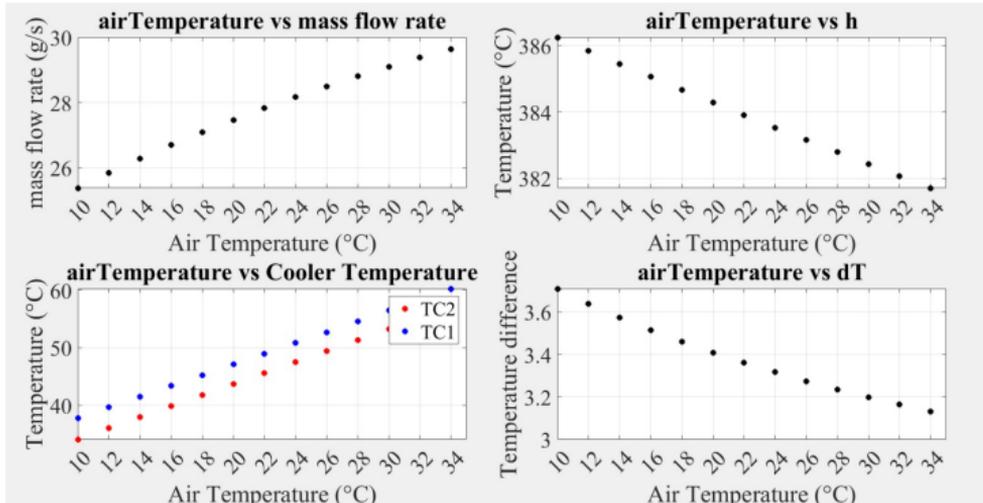
Table: Performance parameter: CPU-time ; test variable: mass flow rate



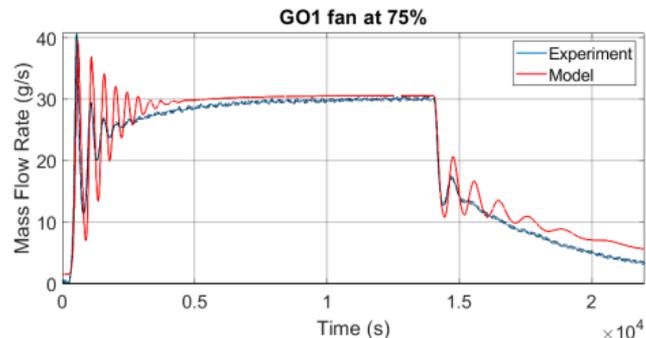
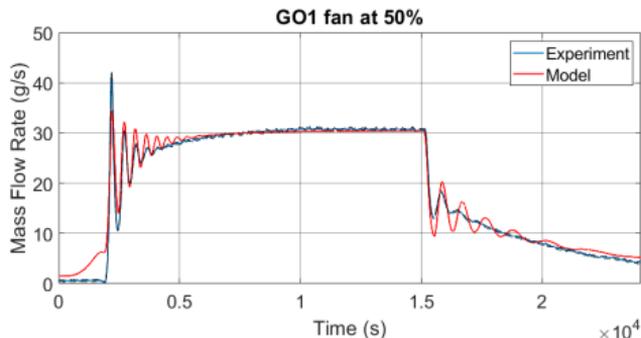
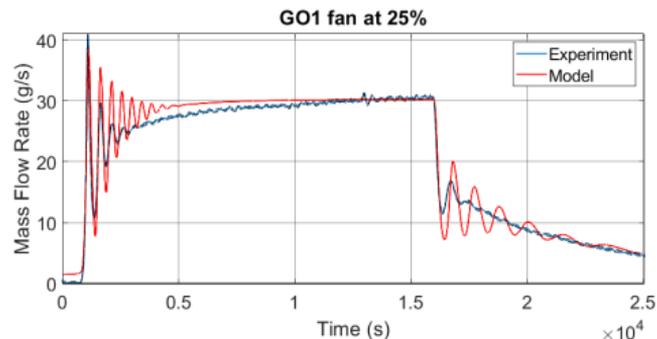
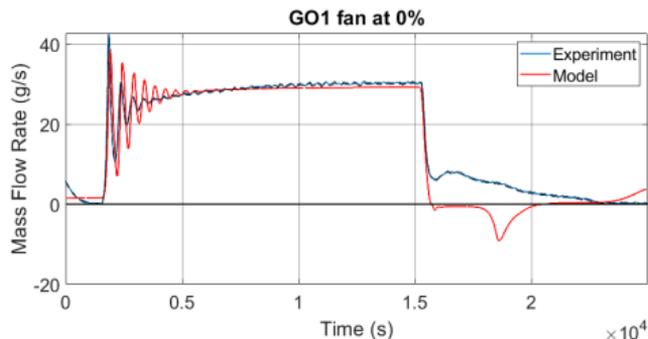
- **Chattering** during simulations (triggering of logic conditions which leads to generation of events and increased CPU time)
- **Radau2a** outperforms all other methods, dampening the oscillations in the initial steps

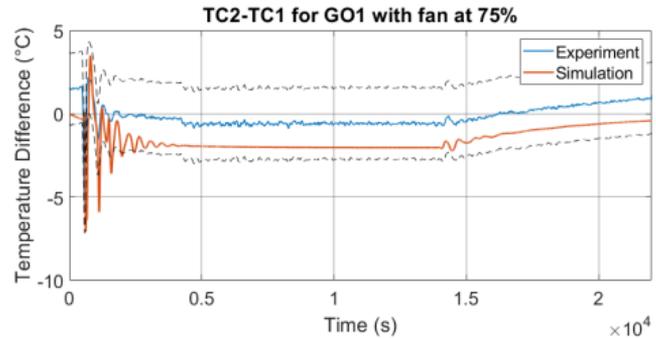
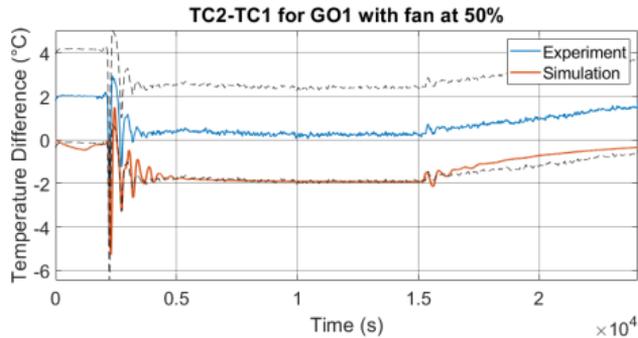
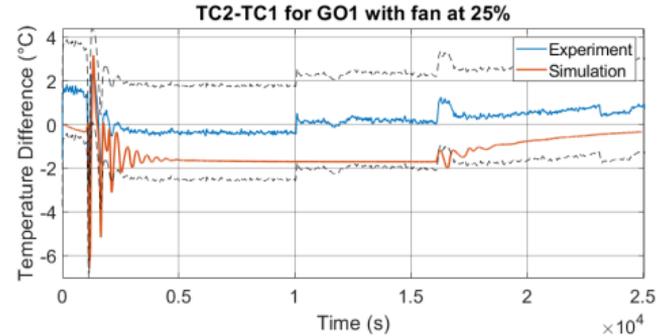
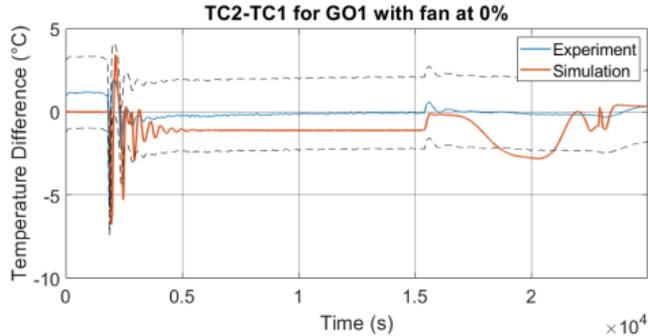


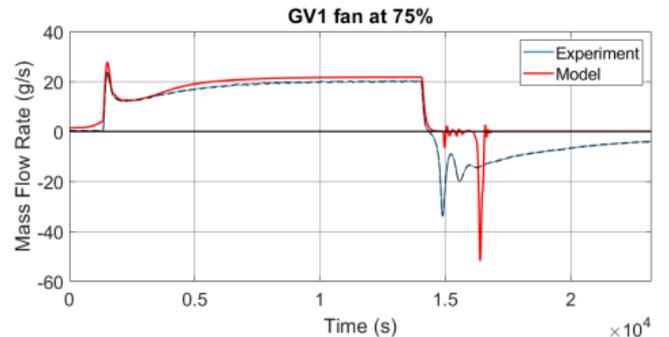
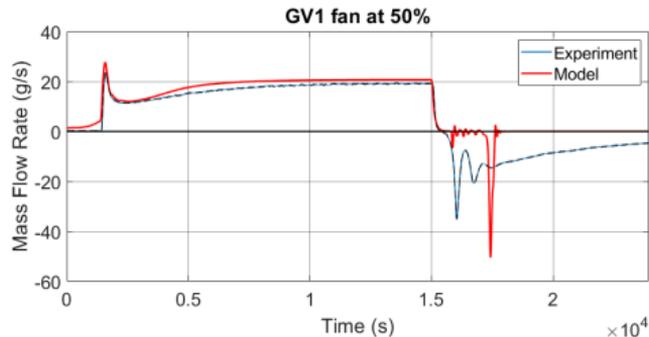
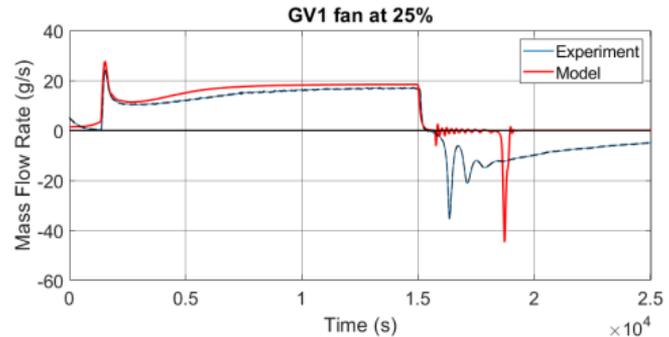
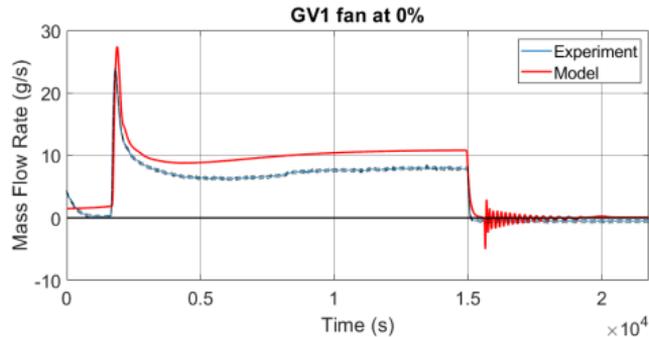
- Sensitivity analysis of the model changing **fan speed** (user-controlled input, constraint for the heat exchange with the environment).
- Understand where the effect of the fan is most visible, characterise the heat transfer in the cooler
- Conditions: air temperature 22 °C, power 450 W (5/8 GV1, 3/8 GV2)

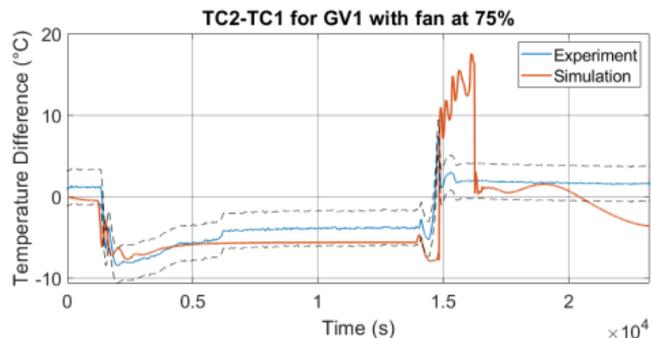
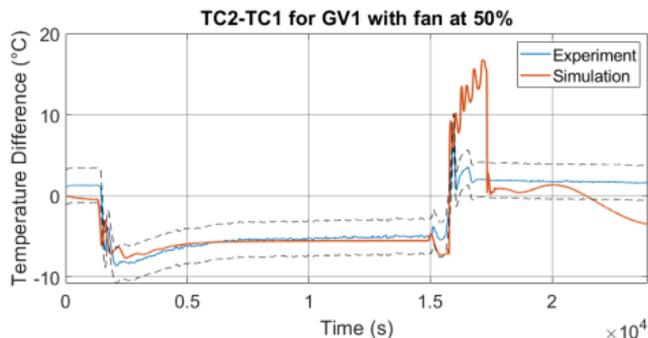
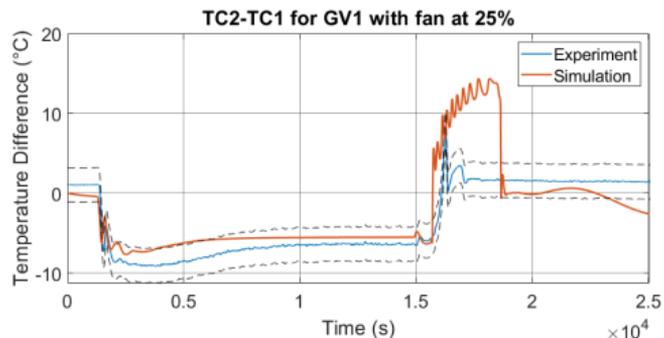
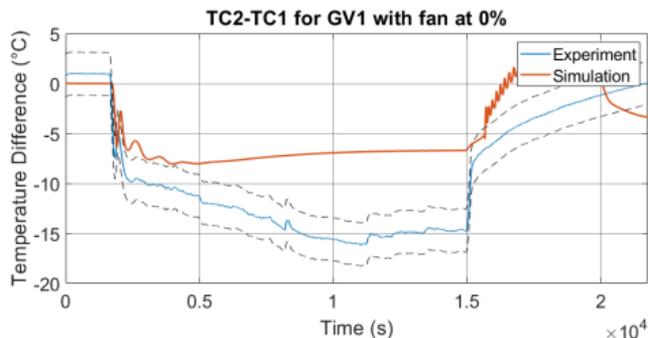


- Sensitivity analysis of the model changing **cooler air temperature** (ambient condition, linked to the heat dissipated by the cooler).
- Conditions: air flow rate  $4 \text{ m}^3\text{s}^{-1}$ , power 450 W (5/8 GV1, 3/8 GV2)









Fan speed	GO1 Cool	GO1 Heat	GV1 Cool	GV1 Heat	GV2 Cool	GV2 Heat
0&	Within	Within	Within	3.77 °C	2.56 °C	Within
25&	0.48 °C	Within	6.11 °C	Within	5.54 °C	0.34 °C
50&	0.16 °C	0.05 °C	5.94 °C	Within	5.65 °C	Within
75&	Within	Within	6.06 °C	Within	6.35 °C	Within

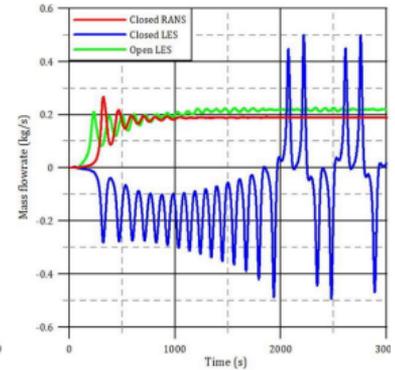
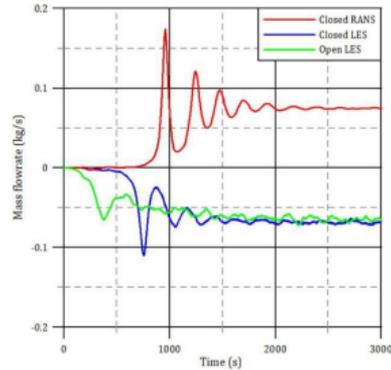
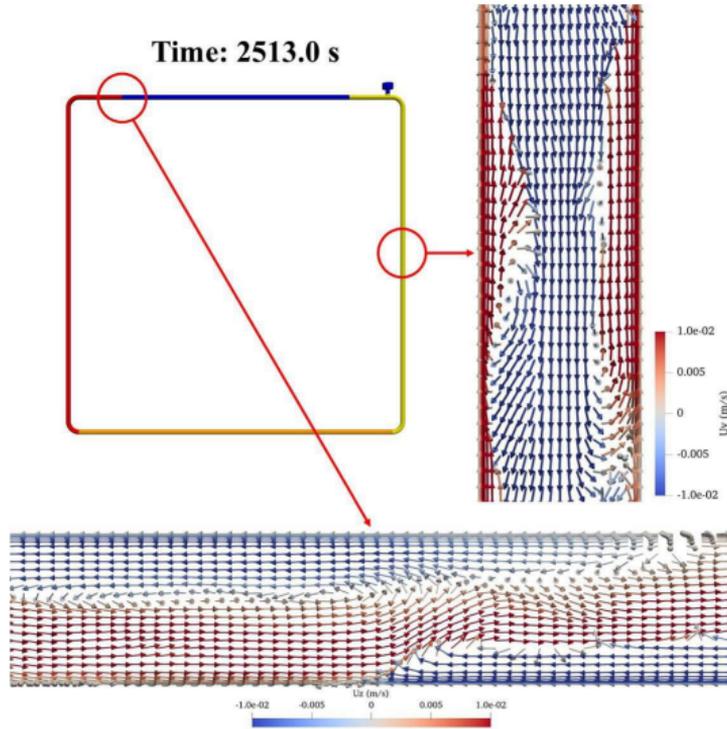
**Table:** Temperature results (error bounds  $\pm 2.15$  °C), square root mean distance between  $TC_2$  and  $TC_1$

Fan speed	GO1 Cool	GO1 Heat	GV1 Cool	GV1 Heat	GV2 Cool	GV2 Heat
0&	129.29%	2.64%	192.31%	10.23%	115.93%	5.63%
25&	16.1%	2.79%	108.24%	7.91%	99.36%	5.86%
50&	14.81%	1.28%	101.27%	6.74%	100.45%	5.64%
75&	39.38%	2.53%	120.67%	5.91%	99.39%	5.39%

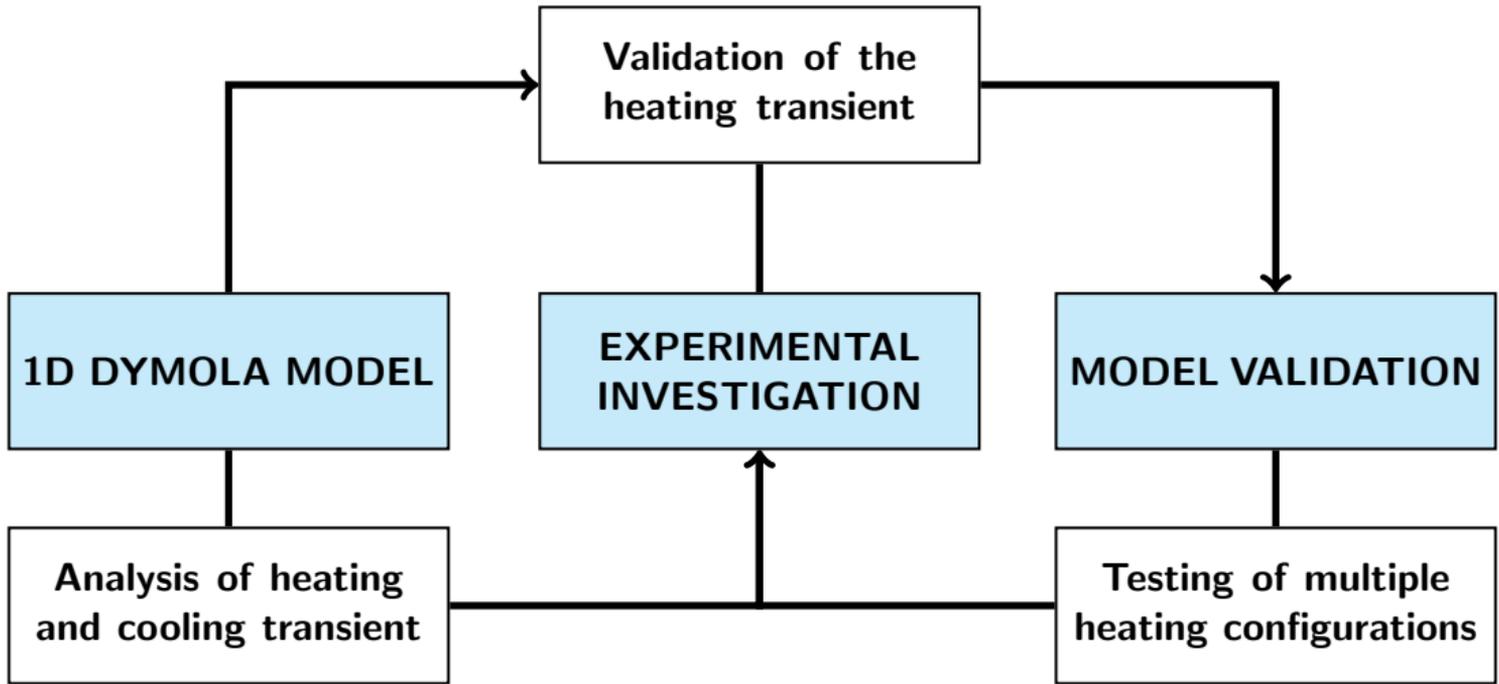
**Table:** Mass flow rate results (error bounds  $\pm 0.2138$  g/s), mean relative error

Possible causes of discrepancy:

- Turbulence phenomena
- Recirculation in the tank region
- Possible boiling near the inner pipe walls
- Non-perfect agreement between initial conditions
- **Note:** the DYMOLA model cannot simulate two-phase flows



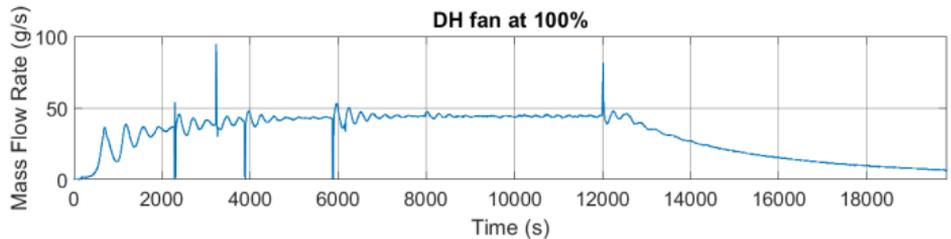
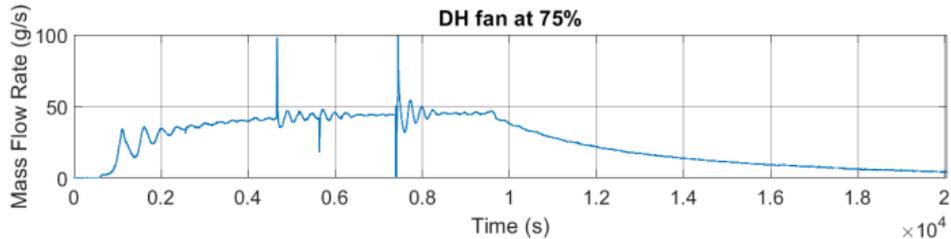
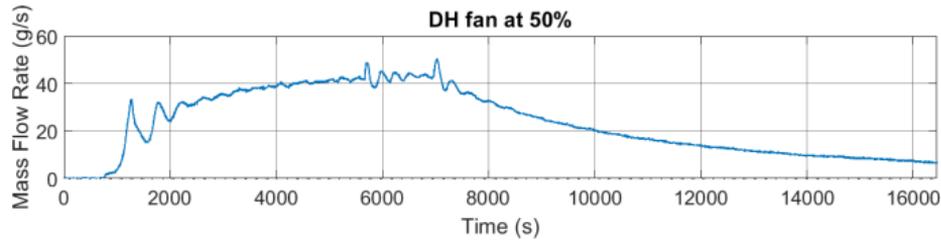
Comparison of mass flow-rate between previous RANS results (POLIMI) and the result of the LES simulations (uniform heating at 1kW and cooler temperature at 180°C, left ; uniform heating at 5.3kW and cooler temperature at 240°C)

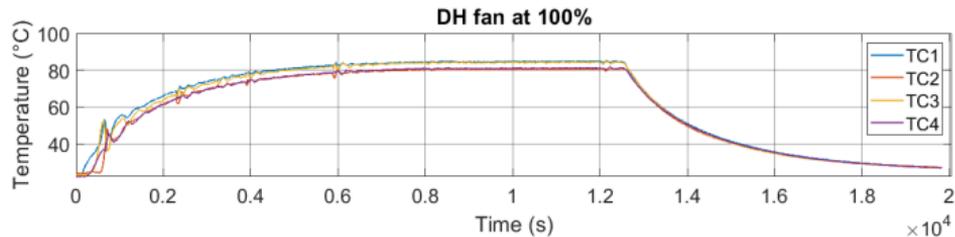
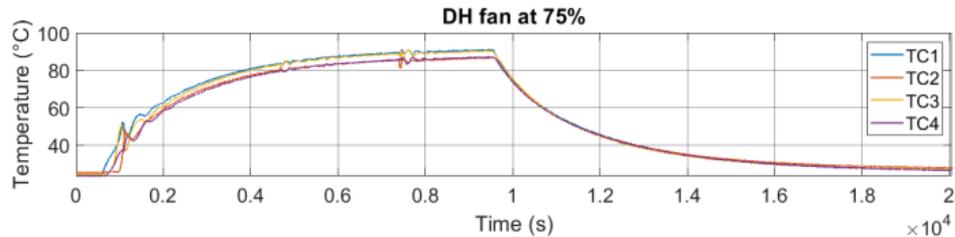
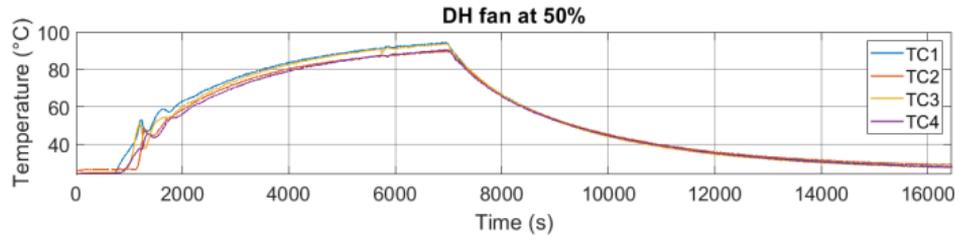


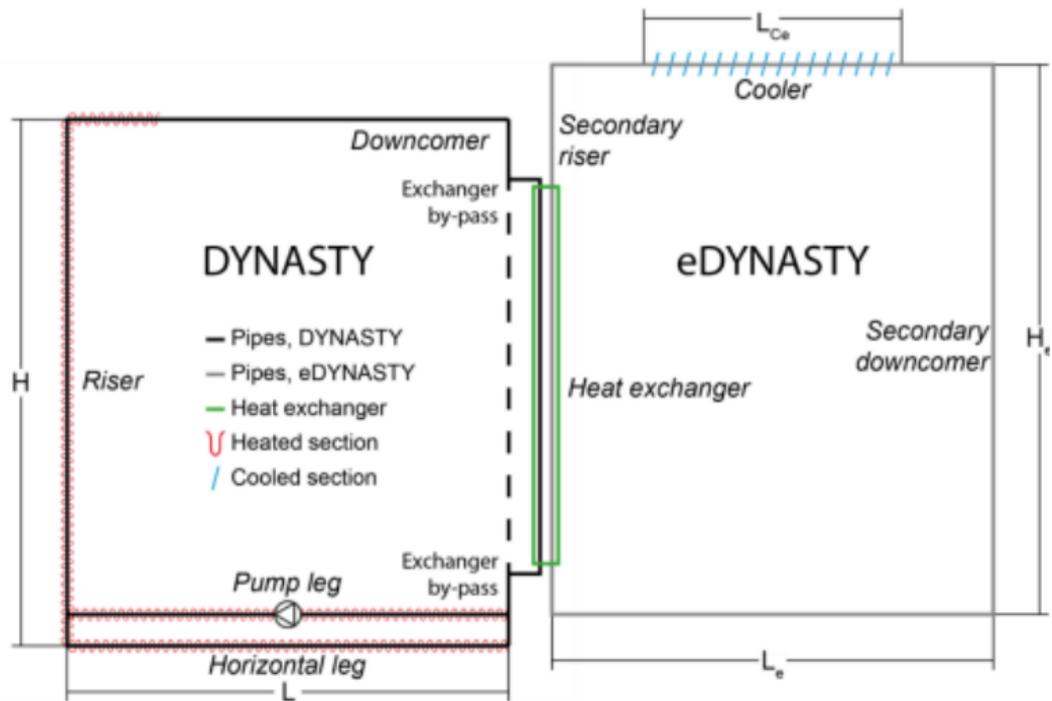
- Installation of the insulant layer
- New validation campaign with both water and glycol
- Inclusion of the feed and discharge tanks in the model
- Realisation of different power transients
- 3D analysis of the cooling transient
- Coupled model with eDYNASTY facility

THANK YOU FOR THE ATTENTION!

DYNASTY CHARACTERISTICS	
Size	Height: 3.09 m Width: 3.10 m Piping: $\varnothing$ 42.16 mm ; thickness 2 mm
Thermal carrier	Water TYFOCOR LS (propylene glycol)
Material	AISI 304/316 L
Heating system	Fibreglass knitted and braided electrical strips (up to 5.3 kW)
Heat exchanger	Finned tube coupled with a cooling fan
Temperature range	20 / 95° C (water)
Pressure	1 atm (filling tank top)







► **DYNASTY-eDYNASTY coupling**

- Pipe-in-pipe heat exchanger
- Internal pipe (DYNASTY) diameter is 38mm
- Annulus pipe (eDYNASTY) diameter is 60mm

